#### Lecture Outline

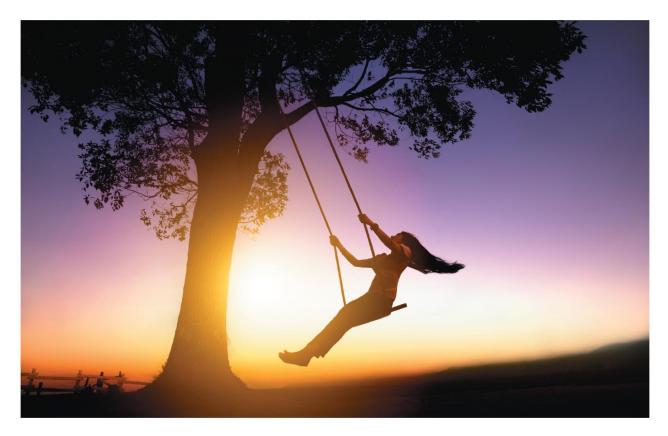
# $\frac{\text{PRINCIPLES & PRACTICE OF}}{PHYSICS}$

# Chapter 15 Periodic Motion

ERIC MAZUR

© 2015 Pearson Education, Inc.

#### **Chapter 15: Periodic Motion**



Chapter Goal: Study the kinematic and dynamics of periodic motion, i.e., motion that repeats itself at regular intervals.

© 2015 Pearson Education, Inc.

#### **Chapter 15 Preview**

Looking Ahead: Fundamental characteristics of periodic motion

- Periodic motion is common at all length scales in the universe:
  - On the atomic level atoms and molecules vibrate in solids.
  - At the human level macroscopic objects, such as guitar strings, vibrate
  - On the cosmological level the universe itself has a repeating pattern of expansion and contraction.

#### **Chapter 15 Preview**

Looking Ahead: Fundamental characteristics of periodic motion

- **Periodic motion** is any motion that repeats itself at regular time intervals. **Oscillation** (or **vibration**) is back-and-forth periodic motion.
- The *period T* is the minimum time interval in which periodic motion repeats, and the **amplitude** *A* of the motion is the magnitude of the maximum displacement of the moving object from its equilibrium position.
- You will learn about the characteristics of periodic motion and the physical variables used to study periodic motion.

## Chapter 15 Preview Looking Ahead: Simple harmonic motion

- Simple harmonic motion is periodic motion in which the displacement of a system from its equilibrium position varies sinusoidally with time. A system moving in this way is called a *simple harmonic oscillator*.
- A restoring force that is linearly proportional to displacement tends to return a simple harmonic oscillator to its equilibrium position. For small displacements, restoring forces are generally proportional to the displacement and therefore cause objects to execute simple harmonic motion about any stable equilibrium position.

## Chapter 15 Preview Looking Ahead: Simple harmonic motion

- A **phasor** is a rotating arrow whose component on a vertical axis traces out simple harmonic motion. The **reference circle** is the circle traced out by the tip of the phasor, and the length of the phasor is equal to the amplitude *A* of the simple harmonic motion.
- You will learn to represent simple harmonic motion both diagramatically and mathematically.

# Chapter 15 Preview Looking Ahead: Fourier series

- Fourier's theorem says that any periodic function with period *T* can be written as a sum of sinusoidal simple harmonic functions of frequency  $f_n = n/T$ , where *n* is an integer. The n = 1 term is the fundamental frequency or first harmonic, and the other components are higher harmonics.
- You will learn how to apply Fourier's theorem to analyze the harmonic content of periodic motion.

# Chapter 15 Preview Looking Ahead: Rotational oscillations

- A horizontal disk suspended at its center by a thin fiber forms a type of *torsional oscillator*.
- A pendulum is any object that swings about a pivot.
   A *simple pendulum* consists of a small object (the bob) attached to a very light wire or rod.
- You will learn how to model the motion of torsional oscillators and simple pendulums mathematically.

# Chapter 15 Preview Looking Ahead: Damped oscillations

- In **damped oscillation**, the amplitude decreases over time due to energy dissipation. The cause of the dissipation is a *damping force* due to friction, air drag, or water drag.
- A damped oscillator that has a large *quality factor Q* keeps oscillating for many periods.
- You will learn how to model the motion of damped oscillators mathematically.

# Chapter 15 Preview Looking Back: Kinematics

- The **non-uniform** motion of an object can be described by the concept of **acceleration**.
- In Chapter 3 you learned how to describe acceleration both graphically and mathematically.

# Chapter 15 Preview Looking Back: Force and motion

- The vector sum of the forces exerted on an object is equal to the acceleration of the object.
- The equation of motion for an object relates the object's acceleration to the vector sum of the forces exerted on it.
- Newton's laws of motion describe the effects forces have on the motion of objects.
- In Chapter 8 you learned how to relate force and motion using Newton's laws.

# Chapter 15 Preview Looking Back: Conservation of energy

- The law of **conservation of energy** states that energy can be transferred from one object to another or converted from one form to another, but it cannot be created or destroyed.
- In Chapter 5 you learned how to identify the types of energy a system has and how to represent the law of conservation of energy mathematically.

# Chapter 15 Preview Looking Back: Angular velocity

- During **rotational motion**, all the particles in an object follow circular paths around the *axis of rotation*.
- The **rotational velocity**  $\omega_{\vartheta}$  of an object is the rate at which the object's rotational coordinate  $\vartheta$  changes.
- The rotational acceleration  $\alpha_{\vartheta}$  is the rate at which an object's rotational velocity changes.
- In Chapter 11 you learned how represent rotational kinematics using diagrams and mathematics.

# Chapter 15 Preview Looking Back: Friction

- When two surfaces touch each other,
  - The component of the contact force normal to the surface is called the **normal force**.
  - The component tangential to the surface is called the **friction force**.
- In Chapter 10, you learned that when surfaces are not moving relative to each other, you have static friction, and when they are moving you have kinetic friction.
- In Chapter 10 you learned how to quantitatively analyze systems experiencing friction.

#### **Chapter 15 Periodic Motion**

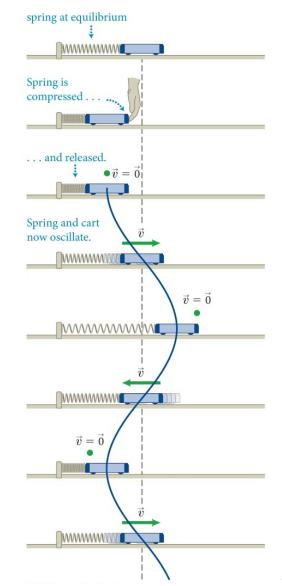
### Concepts

#### **Section Goals**

You will learn to

- Define the concepts of **periodic motion**, **vibration**, and **oscillation**.
- Establish that in a closed system, periodic motion is characterized by the **continuous conversion** between potential energy and kinetic energy.

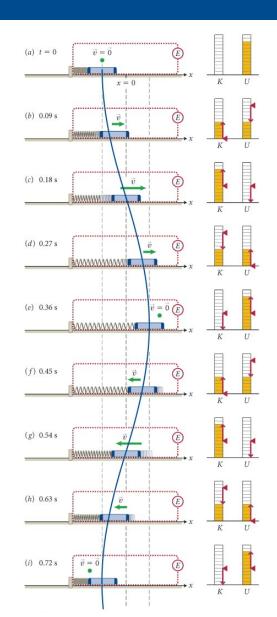
- Any motion that repeats itself at regular time intervals is called **periodic motion.**
- The figure shows the periodic motion of a spring-cart system.
- What forces are present during the oscillation?
- What must be true for the system to oscillate?



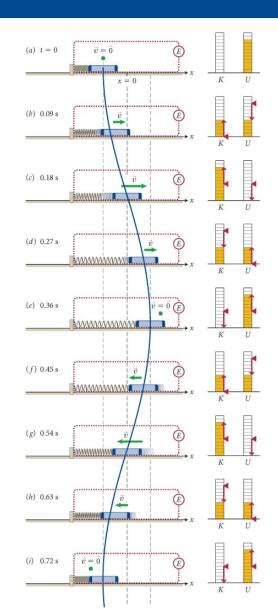
- The time interval it takes to complete a full cycle of the motion is the **period** *T*.
- The inverse of the period is called the **frequency**.

f = 1/T

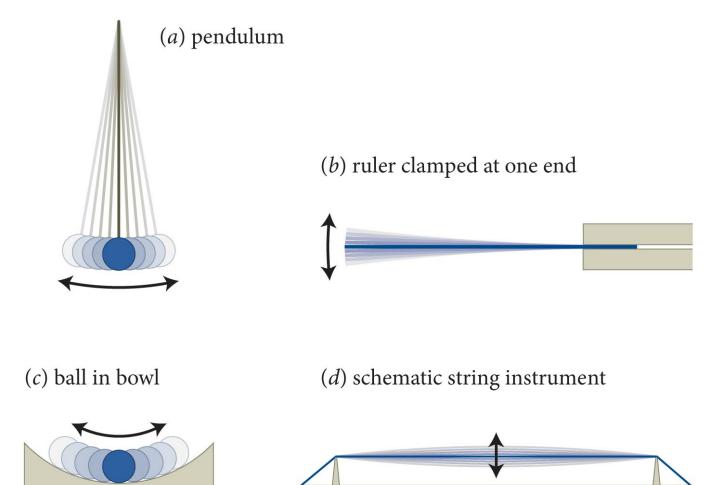
• The object's maximum displacement from the equilibrium position is called the **amplitude** *A*.



- In practice, periodic motion in mechanical systems will die out due to energy dissipation.
- If we ignore these *damping* effects, we find that
  - Periodic motion is characterized by a continuous conversion between potential and kinetic energy in a closed system.

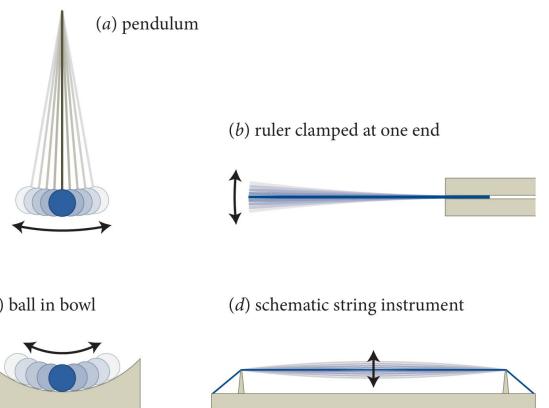


• The figure shows examples of oscillating systems.

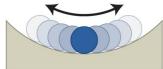


#### Checkpoint 15.3

**15.3** For each system in Figure 15.3, identify (*a*) the restoring force and (b) the type of potential energy associated with the motion.



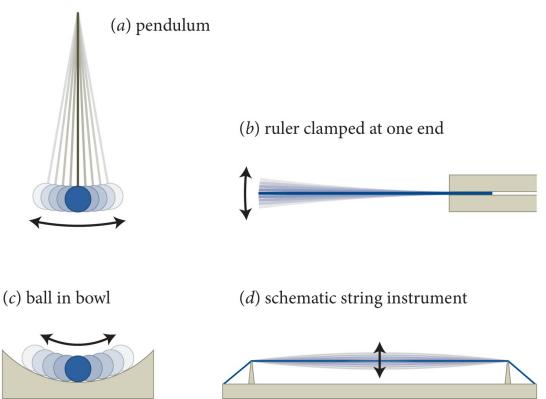
(c) ball in bowl



© 2015 Pearson Education, Inc.

### **Checkpoint 15.3**

- (a) Tangential portion of gravitational force; gravitational potential
  - (b) Vertical component of elastic force in the ruler; elastic potential
  - (c) Tangential component of gravitational force; gravitational potential
  - (d) Vertical component of elastic force in string; elastic potential



## Section 15.1 Question 1

An object hangs motionless from a spring. When the object is pulled down, the sum of the elastic potential energy of the spring and the gravitational potential energy of the object and Earth

- 1. increases.
- 2. stays the same.
- 3. decreases.

## Section 15.1 Question 1

An object hangs motionless from a spring. When the object is pulled down, the sum of the elastic potential energy of the spring and the gravitational potential energy of the object and Earth

#### ✓ 1. increases.

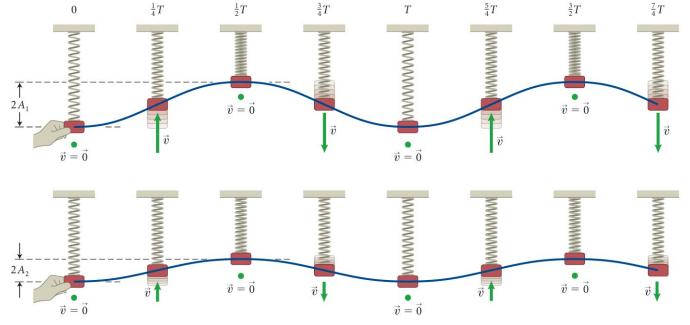
- 2. stays the same.
- 3. decreases.

#### **Section Goals**

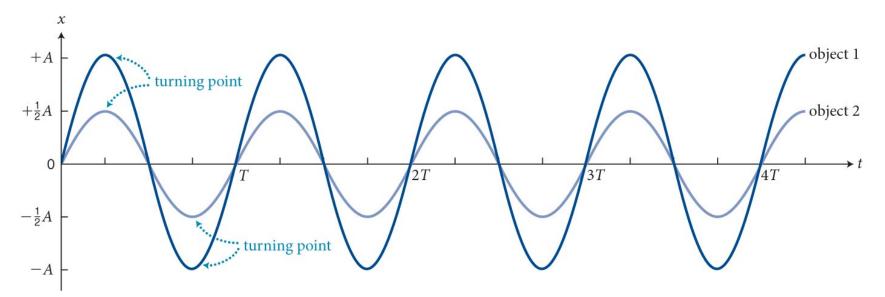
You will learn to

- Define simple harmonic motion and represent it graphically.
- Understand the physical characteristics of the **restoring force** that is responsible for simple harmonic motions.

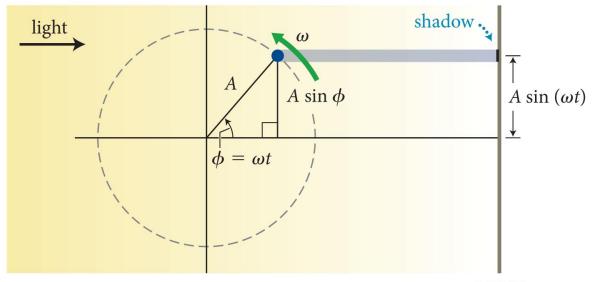
- Investigation of oscillating systems reveal that, when the amplitude is not too large, the period is independent of the amplitude.
- An oscillating system that exhibits this property is called *isochronous*.



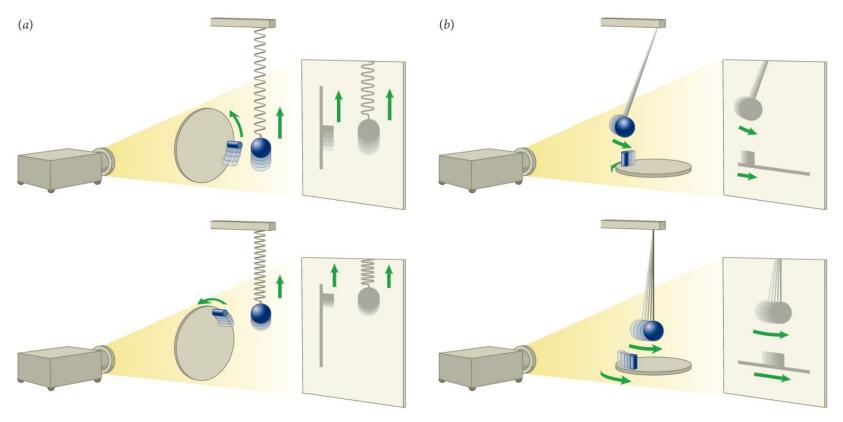
- The x(t) curve of an isochronous oscillation are sinusoidal.
- Periodic motion that yields a sinusoidal x(t) curve is called a simple harmonic motion (SHM):
  - A object executing simple harmonic motion is subject to a linear restoring force that tends to return the object to its equilibrium position and is linearly proportional to the object's displacement from its equilibrium position.



- Simple harmonic motion is closely related to circular motion.
  - The figure shows the shadow of a ball projected onto a screen.
  - As the ball moves in circular motion with constant rotational speed  $\omega$ , the shadow moves with simple harmonic motion.
  - The ball sweeps out at an angle  $\phi = \omega t$  in time *t*.
  - Then the position of the ball's shadow is described by  $A\sin(\omega t)$ , where A is the radius of the circle.



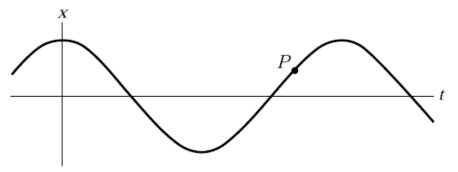
• As illustrated in the figure, the correspondence between circular motion and simple harmonic motion can be demonstrated experimentally.



https://www.youtube.com/watch?v=9r0HexjGRE4

# Section 15.2 Question 2

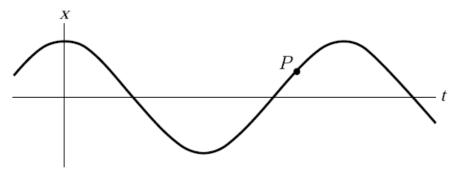
A mass attached to a spring oscillates back and forth as indicated in the position vs. time plot below. At point *P*, the mass has



- 1. positive velocity and positive acceleration.
- 2. positive velocity and negative acceleration.
- 3. positive velocity and zero acceleration.
- 4. negative velocity and positive acceleration.
- 5. negative velocity and negative acceleration.
- 6. negative velocity and zero acceleration.
- 7. zero velocity but is accelerating (positively or negatively).

# Section 15.2 Question 2

A mass attached to a spring oscillates back and forth as indicated in the position vs. time plot below. At point *P*, the mass has



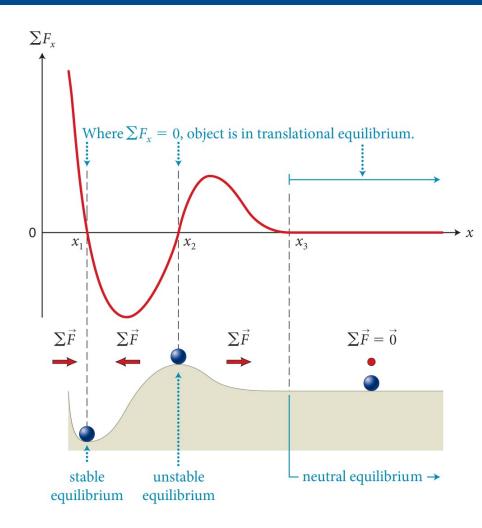
- 1. positive velocity and positive acceleration.
- 2. positive velocity and negative acceleration.
  - 3. positive velocity and zero acceleration.
  - 4. negative velocity and positive acceleration.
  - 5. negative velocity and negative acceleration.
  - 6. negative velocity and zero acceleration.
  - 7. zero velocity but is accelerating (positively or negatively).

#### **Section Goals**

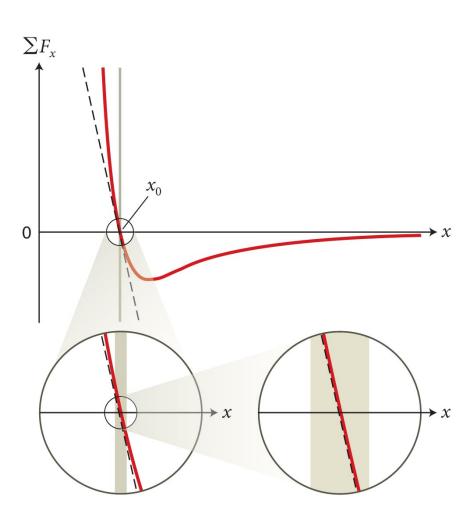
You will learn to

- Correlate the **restoring force** and the resulting motion of a simple harmonic oscillator.
- Relate the period of simple harmonic oscillations to the magnitude of the restoring force.
- Establish that the period of a simple pendulum is independent of the mass of the pendulum.

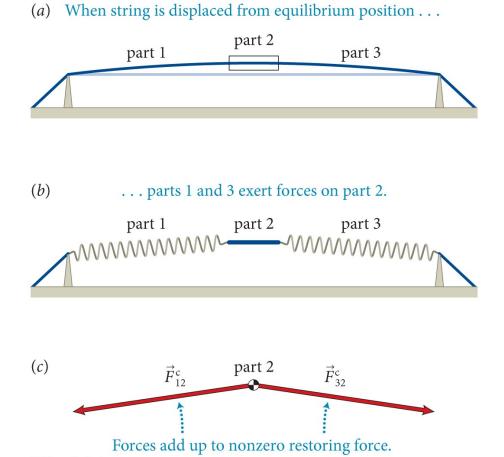
- Periodic motion requires a restoring force that tends to return the object to the equilibrium position.
- A consequence of the restoring force about a stable equilibrium position is
  - In the absence of friction, a small displacement of a system from a position of stable equilibrium causes the system to oscillate.



- As illustrated in the figure:
  - For sufficiently small displacements away from the equilibrium position x<sub>0</sub>, restoring forces are always linearly proportional to the displacement.
  - Consequently, for small displacements, objects execute simple harmonic motion about a stable equilibrium position.



• The figure illustrates the cause for the restoring force for a taut string displaced from its equilibrium position.

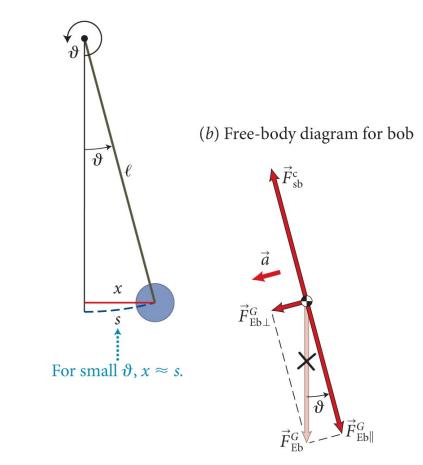


© 2015 Pearson Education, Inc.

- The restoring force for a simple pendulum is provided by the component of the gravitational force perpendicular to the string.
- From the free-body diagram we can see that the magnitude of the restoring force on the bob is

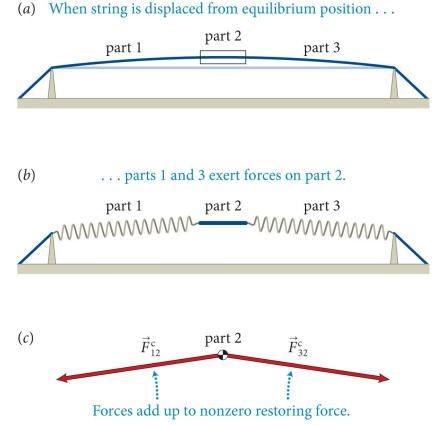
 $mg\sin\vartheta = (mg)(x/\ell) = (mg/\ell)x$ 

(a) Pendulum displaced from equilibrium



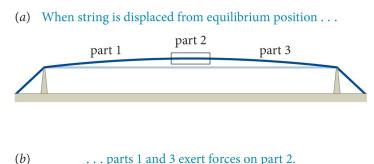
#### **Example 15.1 Displaced string**

Show that for small displacements the restoring force exerted on part 2 of the displaced string in Figure 15.14 is linearly proportional to the displacement of that part from its equilibrium position.



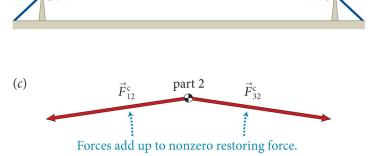
#### Example 15.1 Displaced string (cont.)

**1** GETTING STARTED Figure 15.14*c* shows the forces exerted by parts 1 and 3 on part 2 when the string is displaced from its equilibrium position. I'll assume that these forces are much greater than the force of gravity exerted on part 2 so that I can ignore gravity in this problem. The force that pulls the string away from the equilibrium position is not shown, which means the string has been released after being pulled away from the equilibrium position.



part 1

ΛΛΛΛΛΛΛΛΛΛΛΛΛ

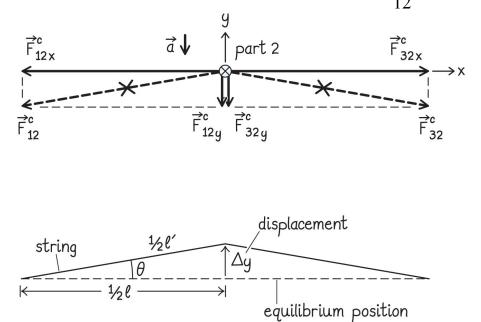


part 2

part 3

#### Example 15.1 Displaced string (cont.)

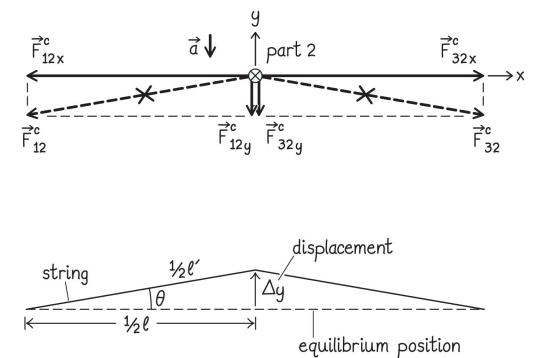
**1** GETTING STARTED I begin by making a free-body diagram and choosing a set of axes (Figure 15.17). The *x* components of the forces  $\vec{F}_{12}^{c}$  and  $\vec{F}_{32}^{c}$  cancel; the sum of the *y* components is the restoring force. The magnitude of the *y* components is determined by the angle  $\theta$  between the *x* axis and either  $\vec{F}_{12}^{c}$  or  $\vec{F}_{22}^{c}$ .



© 2015 Pearson Education, Inc.

#### Example 15.1 Displaced string (cont.)

**①** GETTING STARTED I also make a sketch of the displaced string, showing the displacement  $\Delta y$  of part 2 from its equilibrium position. I denote the length of the string in its equilibrium position by  $\ell$  and the length of the displaced string by  $\ell'$ .



© 2015 Pearson Education, Inc.

#### Example 15.1 Displaced string (cont.)

**2** DEVISE PLAN The forces  $\vec{F}_{12}^{c}$  and  $\vec{F}_{32}^{c}$  are equal in magnitude and their *y* components are determined by sin  $\theta$ , which is equal to  $\Delta y / (\frac{1}{2}\ell')$ . If the displacement is small, I can assume that the length of the string doesn't change much from its equilibrium value, so that  $\ell \approx \ell'$ . Because the forces  $\vec{F}_{12}^{c}$  and  $\vec{F}_{32}^{c}$  are proportional to the tension in the string, I can also consider these forces to be constant. Using this information, I can express the restoring force in terms of the displacement  $\Delta y$ .

#### Example 15.1 Displaced string (cont.)

**③** EXECUTE PLAN From my sketch I see that the restoring force is  $F_{12\nu}^{c} + F_{32\nu}^{c}$ . Because  $\vec{F}_{12}^{c}$  and  $\vec{F}_{32}^{c}$  are equal in magnitude, I can write the sum of the y components as  $2F_{12\nu}^{c} = 2F_{12}^{c}\sin\theta$ . I also know that  $\sin\theta = \Delta y / (\frac{1}{2}\ell') \approx \Delta y / (\frac{1}{2}\ell)$ . Combining these two relationships, I obtain for the magnitude of the restoring force:  $F_{\text{restoring}} = 2F_{12v}^{c} \approx (4F_{12}^{c}/\ell)\Delta y$ . For small displacements, the term in parentheses is constant and so the restoring force is, indeed, proportional to the displacement  $\Delta y$ .

#### Example 15.1 Displaced string (cont.)

**4** EVALUATE RESULT I made two assumptions to derive my answer. The first is that gravity can be ignored. Indeed, taut strings tend to be straight, indicating that gravity (which would make the strings sag) doesn't play an appreciable role. The other assumption I made was that the length of the string doesn't change much when it is displaced from its equilibrium position. This assumption is also justified because the displacement of a string tends to be several orders of magnitude smaller than the string length.

- Another way to look at oscillations is to say
  - Oscillations arise from interplay between inertia and a restoring force.
- Using this interplay between inertia and a restoring force we can predict that
  - The period of an oscillating object increases when its mass is increased and decreases when the magnitude of the restoring force is increased.
- However, this relation does not hold for pendulums:
  - The period of a pendulum is independent of the mass of the pendulum.

#### Section 15.4 Question 4

A child and an adult are on adjacent swings at the playground. Is the adult able to swing in synchrony with the child?

- 1. No, this is impossible because the inertia of the two are different.
- 2. Yes, as long as the lengths of the two swings are adjusted.
- 3. Yes, as long as the lengths of the swings are the same.

#### Section 15.4 Question 4

A child and an adult are on adjacent swings at the playground. Is the adult able to swing in synchrony with the child?

- 1. No, this is impossible because the inertia of the two are different.
- 2. Yes, as long as the lengths of the two swings are adjusted.
- ✓ 3. Yes, as long as the lengths of the swings are the same.

#### **Chapter 15: Periodic Motion**

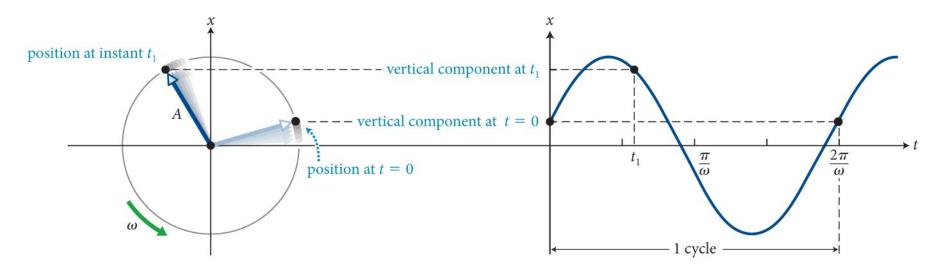
#### **Quantitative Tools**

#### **Section Goals**

You will learn to

- Represent the motion of a simple harmonic oscillator using the **reference circle** and **phasor** diagrams.
- Derive the **kinematic**, **dynamic**, and **energy relationships** for simple harmonic oscillators mathematically.

- A **phasor** is a rotating arrow whose tip traces a circle called the **reference circle**.
- As the phasor rotates counterclockwise at a constant rotational speed  $\omega$ , its vertical component varies sinusoidally and therefore describe a simple harmonic motion.



• If the phasor completes one revolution in a period T, then

$$\omega = \frac{\Delta \vartheta}{\Delta t} = \frac{2\pi}{T}$$

• And the frequency of the corresponding SHM is

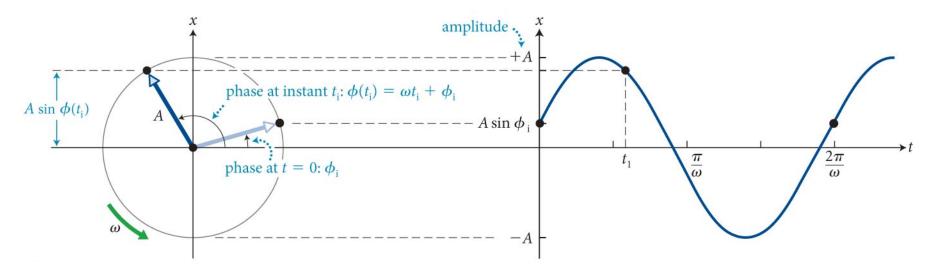
$$J = \frac{T}{T}$$

- The SI units of *f* are  $1 \text{ Hz} \equiv 1 \text{ s}^{-1}$ .
- Combining the previous two equations we get

$$\omega = 2\pi f$$

 ω is often referred to as the angular frequency and has the same unit (s<sup>-1</sup>) as frequency (think: rad/sec)

- The rotational position of the tip of the phasor is called the **phase** of the motion and is given by  $\phi(t) = \omega t + \phi_i$ .
- Then, the vertical component of the phasor can be written as  $x(t) = A \sin \phi(t) = A \sin(\omega t + \phi_i)$  (simple harmonic motion)
- *A* is the amplitude of the phasor and  $\phi_i$  is phase at t = 0 s.
- Means there are **2 boundary conditions**: amplitude & initial phase



• Now, we can obtain the velocity and acceleration of the harmonic oscillator:

$$v_x \equiv \frac{dx}{dt} = \omega A \cos(\omega t + \phi_i) \text{ (simple harmonic motion)}$$

$$a_x = \frac{d^2 x}{dt} = -\omega^2 A \sin(\omega t + \phi_i) \text{ (simple harmonic motion)}$$

$$a_x \equiv \frac{d^2 x}{dt^2} = -\omega^2 A \sin(\omega t + \phi_i)$$
 (simple harmonic motion)

Comparing equations for x(t) and a(t), we can write a<sub>x</sub> = -ω<sup>2</sup>x (simple harmonic motion)
Using Newton's 2<sup>nd</sup> law, ΣF<sub>x</sub> = ma<sub>x</sub>, another constraint!

$$\Sigma F_x = -m\omega^2 x$$
 (simple harmonic motion)

• The work done by the forces exerted on the harmonic oscillator as it moves from the equilibrium position toward the positive *x* direction is

$$W = \int_{x_0}^{x} \sum F_x(x) \, dx = -\int_{x_0}^{x} m\omega^2 x \, dx$$

• This work causes a change in kinetic energy, given by

$$\Delta K = -m\omega^2 \int_{x_0}^x x dx = -m\omega^2 \left[\frac{1}{2}x^2\right]_{x_0}^x = \frac{1}{2}m\omega^2 x_0^2 - \frac{1}{2}m\omega^2 x^2$$

• For a closed system  $\Delta E = \Delta K + \Delta U = 0$ , which gives us

$$\Delta U = U(x) - U(x_0) = \frac{1}{2}m\omega^2 x^2 - \frac{1}{2}m\omega^2 x_0^2$$

• If we let  $U(x_0) = 0$  (a free choice), then

$$E = K + U = \frac{1}{2}mv^{2} + \frac{1}{2}m\omega^{2}x^{2}$$

• Using expressions for x(t) and v(t), we get

$$E = \frac{1}{2}m\omega^2 A^2 \cos^2(\omega t + \phi_i) + \frac{1}{2}m\omega^2 A^2 \sin^2(\omega t + \phi_i)$$
$$= \frac{1}{2}m\omega^2 A^2 \qquad \text{(simple harmonic motion)}$$

• Total energy determined by amplitude and frequency

#### Section 15.5 Question 5

If you know the initial position of an oscillator, what else do you need to know in order to determine the initial phase of the oscillation? Answer all that apply.

- 1. The mass
- 2. The spring constant
- 3. The initial velocity
- 4. The angular frequency
- 5. The amplitude

#### Section 15.5 Question 5

If you know the initial position of an oscillator, what else do you need to know in order to determine the initial phase of the oscillation? Answer all that apply.

- 1. The mass
- 2. The spring constant
- 3. The initial velocity
- 4. The angular frequency
- ✓ 5. The amplitude

 $x(t) = A \sin \phi(t) = A \sin(\omega t + \phi_i)$ If t = 0, need A to get  $\phi_i$ 

#### **Section Goal**

You will learn to

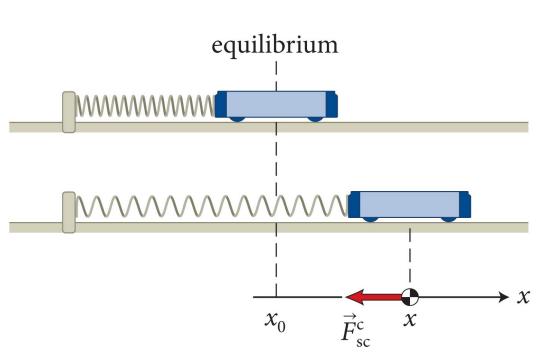
• Apply the mathematical formalism of simple harmonic motion to the case of a **mass attached to a spring**.

- Consider the spring-cart system shown. Let  $x_0 = 0$ .
  - The force exerted by the spring on the cart is

$$F_{SCx}^{C} = -kx$$

• Using  $\Sigma F_x = ma_x$ , we can find the equation of motion for the car to be

$$\frac{d^2x}{dt^2} = -\frac{k}{m}x$$



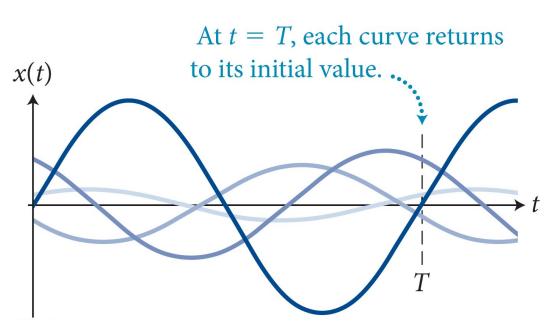
• Comparing the previous equation to Equation 15.10, we get

$$\omega = +\sqrt{\frac{k}{m}}$$

• Therefore, the motion of the cart is given by

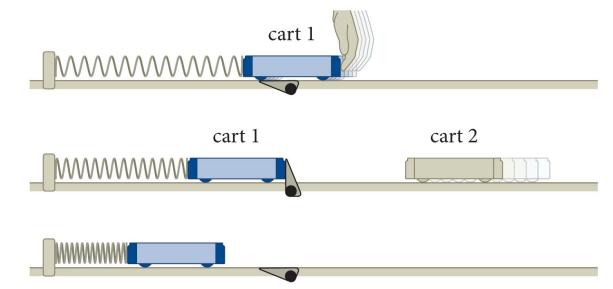
$$x(t) = A \sin\left(\sqrt{\frac{k}{m}t} + \phi_{\rm i}\right)$$

• The figure shows four different solutions that satisfy the equation of motion of the spring-cart system.



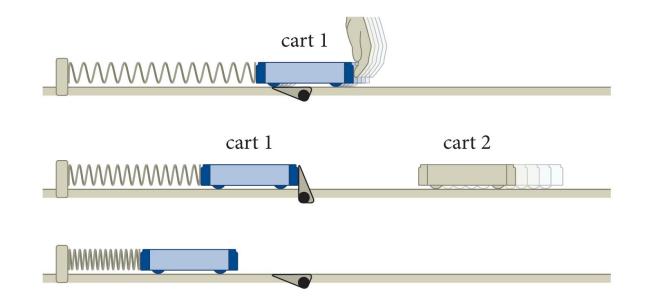
## Example 15.4 Cart stuck with spring already compressed

Cart 1 of mass m = 0.50 kg fastened to a spring of spring constant k = 14 N/m is pushed 15 mm in from its equilibrium position and held in place by a ratchet (Figure 15.27). An identical cart 2 is launched at a speed of 0.10 m/s toward cart 1. The carts collide elastically, releasing the ratchet and setting cart 1 in motion.



# Example 15.4 Cart stuck with spring already compressed (cont.)

After the collision, cart 2 is immediately removed from the track. (*a*) What is the maximum compression of the spring? (*b*) How many seconds elapse between the instant the carts collide and the instant the spring reaches maximum compression?

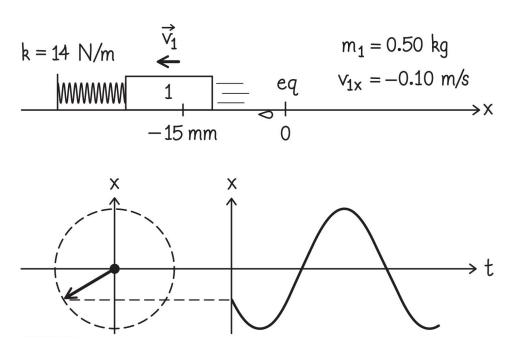


## Example 15.4 Cart stuck with spring already compressed (cont.)

• GETTING STARTED If I ignore the effect of the spring during the collision, I can say that the elastic collision interchanges the velocities of the two carts. I make a sketch of the initial condition of cart 1 just before the collision, choosing the positive *x* axis to the right, the equilibrium position at x = 0, and the initial displacement of the cart 15 mm to the left of x = 0.

# Example 15.4 Cart stuck with spring already compressed (cont.)

• GETTING STARTED I also draw a reference circle and sketch the oscillation resulting from the collision (Figure 15.28). With this choice of axis, the *x* component of the initial velocity of cart 1 is  $v_{x,i} = -0.10$  m/s. Given *m* and *k*, and  $\omega^2 = k/m$ ,  $\omega$ = 5.3 s<sup>-1</sup>.



# Example 15.4 Cart stuck with spring already compressed (cont.)

• GETTING STARTED In contrast to the situation in Example 15.3, the initial displacement of cart 1 is not equal to the amplitude of the oscillation because the collision increases the cart's displacement from the equilibrium position. In other words, cart 1 moves leftward immediately after the collision. It continues moving in this direction until the elastic restoring force building up in the compressing spring causes the cart to stop. This maximum-compression position gives the amplitude of the oscillation.

## Example 15.4 Cart stuck with spring already compressed (cont.)

**2** DEVISE PLAN I can determine the value of the amplitude from the mechanical energy of the cart-spring system, which is the sum of the initial potential energy in the spring and the initial kinetic energy of the cart. The potential energy in the spring is given by Eq. 9.23; because the equilibrium position  $x_0$  is at the origin, this equation reduces to  $U_{\text{spring}} = \frac{1}{2}kx^2$ .

## Example 15.4 Cart stuck with spring already compressed (cont.)

**2** DEVISE PLAN The kinetic energy is given by  $K = \frac{1}{2}mv^2$ . At the position of maximum compression, all of the mechanical energy is stored in the spring, x = -A, and so  $E_{\text{mech}} = U_{\text{spring}} = \frac{1}{2}kA^2$ . Once I know A, I can determine the initial phase from Eq. 15.6. I can then use that same equation to solve for t at the position of maximum compression, where x = -A.

## Example 15.4 Cart stuck with spring already compressed (cont.)

SEXECUTE PLAN (*a*) The initial kinetic and potential energies of the cart-spring system are

$$K = \frac{1}{2}(0.50 \text{ kg})(-0.10 \text{ m/s})^2 = 0.0025 \text{ J}$$

$$U = \frac{1}{2}kx^{2} = \frac{1}{2}(14 \text{ N/m})(-15 \text{ mm})^{2} \left(\frac{1.0 \text{ m}}{1000 \text{ mm}}\right)^{2} = 0.0016 \text{ J}$$
  
So

$$E_{\text{mech}} = K + U = (0.0025 \text{ J}) + (0.0016 \text{ J}) = 0.0041 \text{ J}.$$

## Example 15.4 Cart stuck with spring already compressed (cont.)

**③** EXECUTE PLAN At the position of maximum compression, all of this energy is stored in the spring, and so  $\frac{1}{2}kA^2 = 0.0041$  J, or with the *k* value given,

$$A = \sqrt{\frac{2(0.0041 \text{ J})}{14 \text{ N/m}}} = 0.024 \text{ m} = 24 \text{ mm.}$$

## Example 15.4 Cart stuck with spring already compressed (cont.)

SEXECUTE PLAN (*b*) Substituting the value for *A* determined in part *a* and the initial condition  $x_i = -15$  mm at t = 0 into Eq. 15.6, I obtain

$$x(0) = A \sin(0 + \phi_i) = (24 \text{ mm}) \sin \phi_i = -15 \text{ mm}$$

$$\sin\phi_{i} = \frac{-15\text{mm}}{24\text{mm}} = -0.63 \text{ or } \phi_{i} = \sin^{-1}(-0.63).$$

## Example 15.4 Cart stuck with spring already compressed (cont.)

**3** EXECUTE PLAN Two initial phases satisfy this relationship,  $\phi_i = -0.68$  and  $\phi_i = -\pi + 0.68 = -2.5$ , but only the latter gives a negative *x* component of the velocity (see Eq. 15.7), as required by the initial condition.

At the first instant of maximum compression,  $\sin(\omega t + \phi_i) = -1$ , which means  $\omega t = \phi_i = -\frac{1}{2}\pi$ . Solving for *t* yields  $t = (-\frac{1}{2}\pi - \phi_i)/\omega = [-\frac{1}{2}\pi - (-2.5)]/(5.3 \text{ s}^{-1}) = 0.17 \text{ s}.$ 

## Example 15.4 Cart stuck with spring already compressed (cont.)

Devaluate Result At 24 mm, the amplitude is greater than the 15-mm initial displacement from the equilibrium position, as I would expect. From the reference circle part of my sketch I see that it takes about one-eighth of a cycle to go from the position of impact to the position of maximum compression.

## Example 15.4 Cart stuck with spring already compressed (cont.)

EVALUATE RESULT From Eq. 15.1 I see that the number of seconds needed to complete one cycle is  $T = 2\pi/\omega = 2\pi/(5.3 \text{ s}^{-1}) = 1.2 \text{ s}$ , and so the 0.17-s value I obtained for seconds elapsed between collision and maximum compression is indeed close to one-eighth of a cycle.

# Example 15.4 Cart stuck with spring already compressed (cont.)

EVALUATE RESULT The assumption that the influence of the spring can be ignored during the collision is justified because the force exerted by the spring is small relative to the force of the impact: At a compression of 15 mm, the magnitude of the force exerted by the spring is (14 N/m)(0.015 m) = 0.21 N.

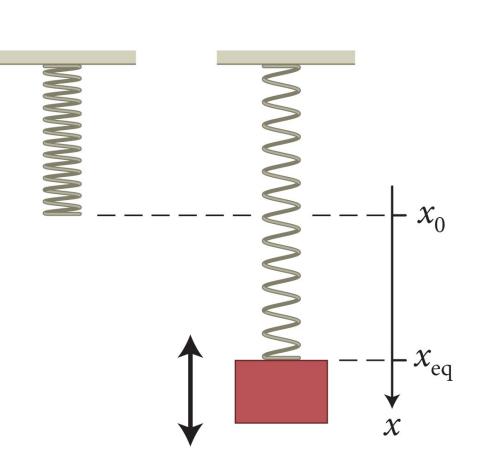
# Example 15.4 Cart stuck with spring already compressed (cont.)

• EVALUATE RESULT The force of impact is given by the time rate of change in the cart's momentum,  $\Delta \vec{p} / \Delta t$ . The magnitude of the momentum change is  $\Delta p = m\Delta v = (0.50 \text{ kg})(0.10 \text{ m/s}) = 0.050 \text{ kg} \cdot \text{m/s}$ . If the collision takes place in, say, 20 ms, the magnitude of the force of impact is  $(0.050 \text{ kg} \cdot \text{m/s})/(0.020 \text{ s}) = 2.5 \text{ N}$ , which is more than 10 times greater than the magnitude of the force exerted by the spring.

(a)

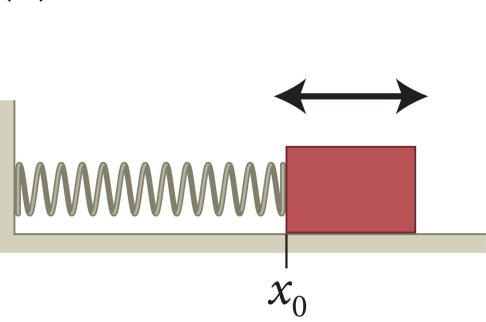
### **Example 15.5 Vertical oscillations**

A block of mass m = 0.50kg is suspended from a spring of spring constant k = 100 N/m. (a) How far below the end of the relaxed spring at  $x_0$  is the equilibrium position  $x_{eq}$ of the suspended block (Figure 15.29*a*)?



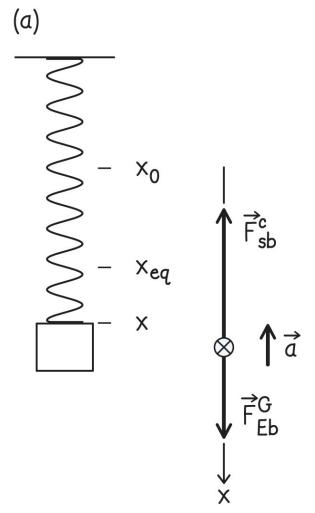
### **Example 15.5 Vertical oscillations (cont.)**

(b) Is the frequency f with (b) which the block oscillates about this equilibrium position  $x_{eq}$  greater than, smaller than, or equal to that of an identical system that oscillates horizontally about  $x_0$  on a surface for which friction can be ignored (Figure 15.29*b*)?



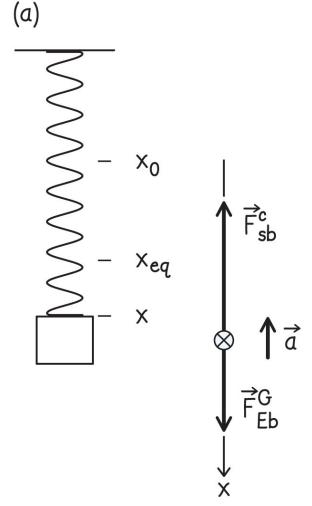
### **Example 15.5 Vertical oscillations (cont.)**

• GETTING STARTED I begin by making a free-body diagram for the suspended block, choosing the positive x axis pointing downward. Two forces are exerted on the block: an upward force  $\vec{F}_{sh}^{c}$ exerted by the spring and a downward gravitational force  $\vec{F}_{\rm Eb}^{G}$ exerted by Earth (Figure 15.30*a*).



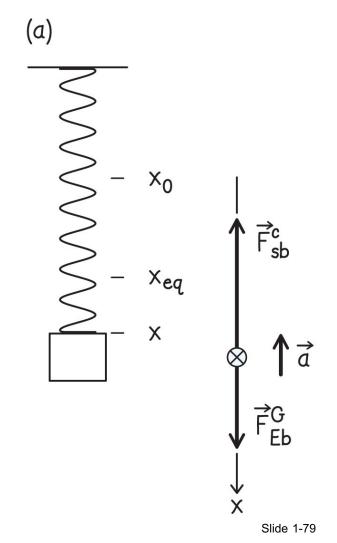
### **Example 15.5 Vertical oscillations (cont.)**

• GETTING STARTED When the suspended block is in translational equilibrium at  $x_{eq}$ (which lies below  $x_0$ ), the vector sum of these forces must be zero. With the block at  $x_{eq}$ , the spring is stretched such that the end attached to the block is also at  $x_{eq}$ .



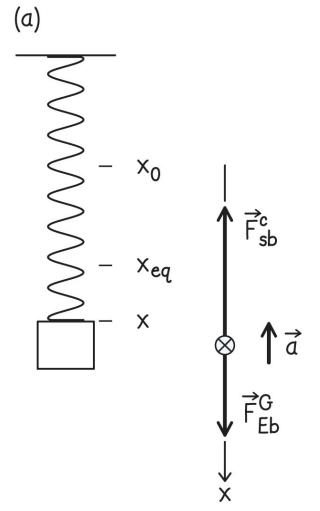
### **Example 15.5 Vertical oscillations (cont.)**

• GETTING STARTED When the block is below  $x_{eq}$ , the spring is stretched farther, causing the magnitude of  $\vec{F}_{sb}^{c}$  to increase, and so now the vector sum of the forces exerted on the block points upward.



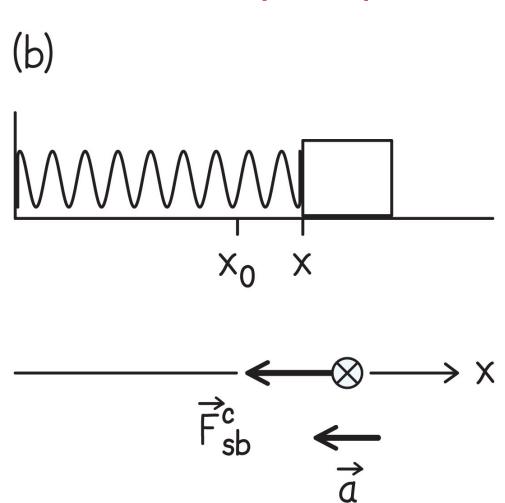
### **Example 15.5 Vertical oscillations (cont.)**

**①** GETTING STARTED When the block is above  $x_{eq}$  (but below the position  $x_0$  of the end of the spring when it is relaxed), the spring is stretched less than when the block is at  $x_{eq}$ , causing the magnitude of  $\vec{F}_{sb}^{c}$  to decrease, and so the vector sum of the forces exerted on the block points downward. The vector sum of  $\vec{F}_{ch}^{c}$  and  $\vec{F}_{\rm Eh}^{G}$  thus serves as a restoring force.



### **Example 15.5 Vertical oscillations (cont.)**

**1** GETTING STARTED I also make a free-body diagram for the horizontal arrangement (Figure 15.30b), showing only the horizontal forces (the force of gravity and the normal force exerted by the surface cancel out). I let the positive x axis point to the right. In this case, the restoring force is  $\vec{F}_{sb}^{c}$  only.



## **Example 15.5 Vertical oscillations (cont.)**

**2** DEVISE PLAN In translational equilibrium, the vector sum of the forces exerted on the suspended block is zero, and so I can determine the magnitude of the force exerted by the spring. I can then use Hooke's law (Eq. 8.20) to determine the distance between the equilibrium position and  $x_0$ . To compare the oscillation frequencies of the two systems, I should write the simple harmonic oscillator equation for each system in the form given by Eq. 15.12.

### **Example 15.5 Vertical oscillations (cont.)**

**3** EXECUTE PLAN (*a*) For translational equilibrium, I have

$$\Sigma F_{x} = F_{sbx}^{c} + F_{Ebx}^{G} = -k(x_{eq} - x_{0}) + mg = 0,$$

where  $x_{eq} - x_0$  is the displacement of the spring's end from its relaxed position. Solving for  $x_{eq} - x_0$ , I obtain

$$x_{eq} - x_0 = \frac{mg}{k} = \frac{(0.50 \text{ kg})(9.8 \text{ m/s}^2)}{100 \text{ N/m}} = 0.049 \text{ m}.\checkmark$$

### **Example 15.5 Vertical oscillations (cont.)**

**3** EXECUTE PLAN (*b*) For the horizontal arrangement, I have in the situation depicted in my sketch (Figure 15.30*b*)

$$\Sigma F_{x} = F_{sbx}^{c} = -k(x - x_{0})$$
 (1)

so if I let the origin of my x axis be at the position of the end of the relaxed spring,  $x_0 = 0$ , the rightmost factor in Eq. 1 reduces to -kx and thus  $\Sigma F_x = -kx$ .

### **Example 15.5 Vertical oscillations (cont.)**

**3** EXECUTE PLAN Next I turn to the vertical arrangement. In the position illustrated in Figure 15.30*a*, the *x* component of the upward force exerted by the spring is

$$F_{sbx}^{c} = -k(x - x_{0}) = -k(x - x_{eq}) - k(x_{eq} - x_{0}). \quad (2)$$

### **Example 15.5 Vertical oscillations (cont.)**

**3** EXECUTE PLAN From part *a* I know that  $k(x_{eq} - x_0)$  is equal to *mg*. The *x* component of the vector sum of the forces exerted on the block at position *x* is thus

$$\Sigma F_x = F_{Ebx}^G + F_{sbx}^c = mg - k(x - x_{eq}) - mg$$
$$= -k(x - x_{eq}).$$

### **Example 15.5 Vertical oscillations (cont.)**

**③** EXECUTE PLAN If, as usual, I let the origin be at the equilibrium position,  $x_{eq} = 0$ , then this result for  $\Sigma F_x$  is identical to Eq. 1. Comparing these results to Eq. 15.12, I see that in both cases  $k = m\omega^2$  and so the oscillation frequencies  $f = \omega/2\pi$  of the two systems are the same:  $f_{vert} = f_{hor}$ .

## **Example 15.5 Vertical oscillations (cont.)**

**4** EVALUATE RESULT The two oscillations take place about different equilibrium positions ( $x_0$  for the horizontal case,  $x_{eq}$  for the vertical case), but the effect of the combined gravitational and elastic forces in the vertical arrangement is the same as that of just the elastic force in the horizontal arrangement because the force exerted by the spring is linear in the displacement.

## Section 15.6 Question 6

If you know the mass of an object hanging from a spring in an oscillating system, what else do you need to know to determine the period of the motion? Answer all that apply.

- 1. The spring constant
- 2. The initial velocity
- 3. The angular frequency
- 4. The amplitude

## Section 15.6 Question 6

If you know the mass of an object hanging from a spring in an oscillating system, what else do you need to know to determine the period of the motion? Answer all that apply.

✓ 1. The spring constant

- 2. The initial velocity
- 3. The angular frequency
- 4. The amplitude

## **Section Goals**

You will learn to

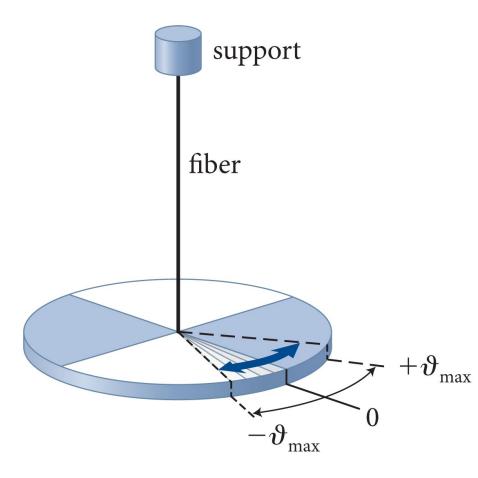
- Extend the concept of simple harmonic motion to rotational situations involving **torsional oscillators** and **simple pendulums**.
- Apply **conservation of momentum** to situations involving relativistic objects.

- Some simple harmonic oscillators involve rotational motion.
  - The torsion oscillator shown is an example of this type of oscillator.
  - The equation of motion for the disk is

$$\Sigma \tau_{\vartheta} = I \alpha_{\vartheta}$$

• For small rotational displacements,

$$\boldsymbol{\tau}_{\vartheta} = -\boldsymbol{\kappa}(\vartheta - \vartheta_{\scriptscriptstyle 0})$$



• If 
$$\vartheta_0 = 0$$
, we get

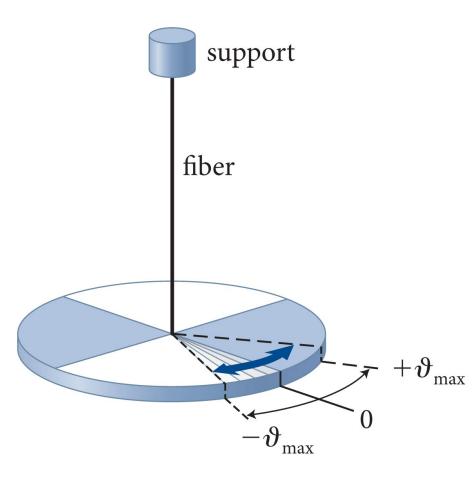
$$\frac{d^2\vartheta}{dt^2} = -\frac{\kappa}{I}\vartheta$$

• Comparing this equation to Equation 15.21, we can write

$$\vartheta = \vartheta_{\max} \sin(\omega t + \phi_{i})$$

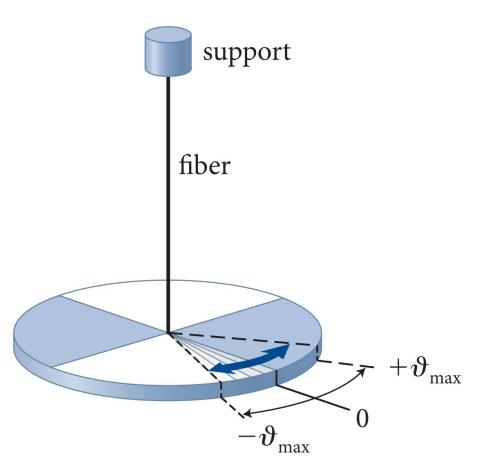
$$\omega = \sqrt{\frac{\kappa}{I}}$$
 (torsional oscillator)

where  $\vartheta_{\text{max}}$  is the maximum rotational displacement.



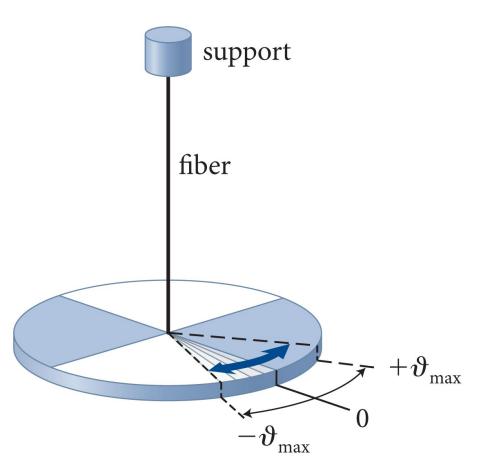
## Checkpoint 15.15

**15.15** For the torsional oscillator shown in Figure 15.31, what effect, if any, does a decrease in the radius of the disk have on the oscillation frequency f? Assume the disk's mass is kept the same.



## Checkpoint 15.15

**15.15** For the torsional oscillator shown in Figure 15.31, what effect, if any, does a decrease in the radius of the disk have on the oscillation frequency f? Assume the disk's mass is kept the same.



Decreasing the radius reduces its rotational inertia (rotates more easily).

If I decreases,  $\omega$  increases, so f increases as well

- The pendulum is another example of a rotational oscillator.
  - The torque caused by the force of gravity about the axis is

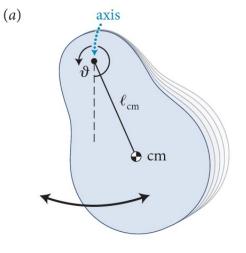
 $\tau_{\vartheta} = -\ell_{\rm cm} \,(mg\,\sin\vartheta)$ 

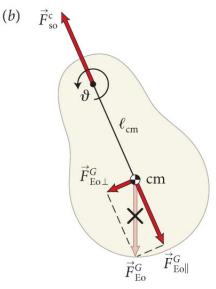
• For small rotational displacements, sin  $\vartheta \approx \vartheta$ 

$$\tau_{\vartheta} = -(m\ell_{\rm cm}g) \vartheta$$

• Using  $\tau_{\vartheta} = I\alpha_{\vartheta} = Id^2 \vartheta/dt^2$ , we get

$$\frac{d^{2}\vartheta}{dt^{2}} = -\frac{m\ell_{\rm cm}g}{I}\vartheta$$
$$\omega = \sqrt{\frac{m\ell_{\rm cm}g}{I}} \text{ (pendulum)}$$



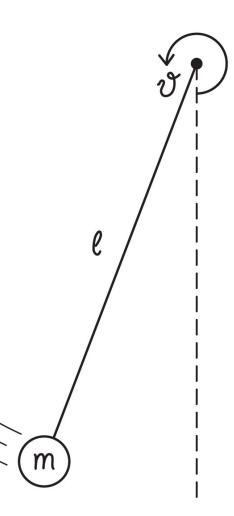


### **Example 15.6: The simple pendulum**

Suppose a simple pendulum consisting of a bob of mass m suspended from a string of length  $\ell$  is pulled back and released. What is the period of oscillation of the bob?

### Example 15.6: The simple pendulum (cont.)

• GETTING STARTED I begin by making a sketch of the simple pendulum (Figure 15.33), indicating the equilibrium position by a vertical dashed line.



### Example 15.6: The simple pendulum (cont.)

**2** DEVISE PLAN The period of the pendulum is related to the angular frequency (Eq. 15.1). To calculate the angular frequency, I can use Eq. 15.33 with  $\ell_{cm} = \ell$ . If I treat the bob as a particle, I can use Eq. 11.30,  $I = mr^2$ , with  $r = \ell$  to calculate the bob's rotational inertia about the point of suspension.

## Example 15.6: The simple pendulum (cont.)

BEXECUTE PLAN Substituting the bob's rotational inertia into Eq. 15.33, I get

$$\omega = \sqrt{\frac{m\ell g}{m\ell^2}} = \sqrt{\frac{g}{\ell}},$$

so, from Eq. 15.1,  $\omega = 2\pi/T$ , I obtain

$$T = \frac{2\pi}{\omega} = 2\pi \sqrt{\frac{\ell}{g}}. \checkmark$$

### Example 15.6: The simple pendulum (cont.)

**4** EVALUATE RESULT My result says that *T* is independent of the mass *m* of the bob, in agreement with what is stated in Section 15.4. Increasing gdecreases the period as it should: A greater g means a greater restoring force, and so the bob is pulled back to the equilibrium position faster. It also makes sense that increasing  $\ell$  increases the period: As my sketch shows, for greater  $\ell$  the bob has to move a greater distance to return to the equilibrium position.

## **Section Goals**

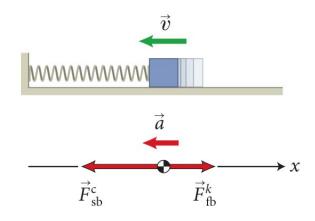
You will learn to

- Integrate the concept of **friction** to oscillatory motion and represent the combined effects graphically.
- Model damped harmonic motion **mathematically**.
- Define the **time constant** for damped harmonic motion.

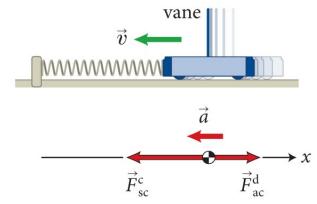
- Mechanical oscillators always involve some friction that causes the energy of the oscillator to convert to thermal energy.
- This will cause the oscillator to slow down.
- Such a system is said to execute a **damped** oscillation.
- The figure shows examples of two damped oscillations.

© 2015 Pearson Education, Inc.

(a) Oscillating block is slowed by friction



(b) Oscillating cart is slowed by air drag on vane



Slide 1-103

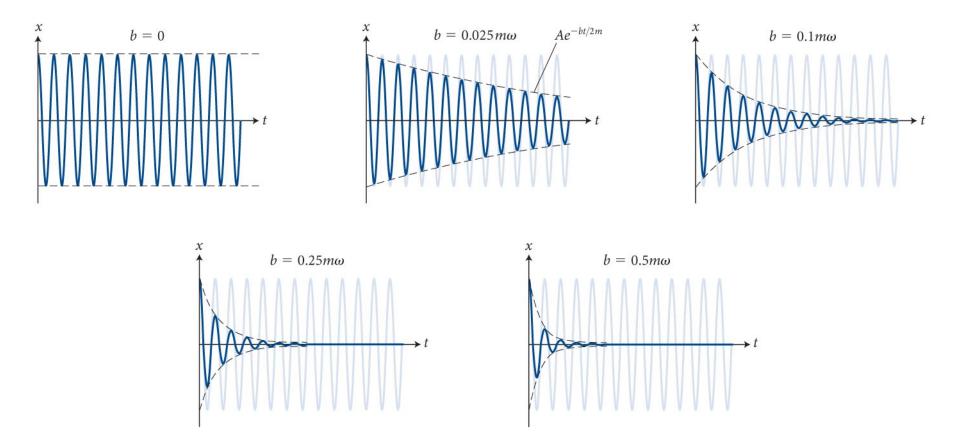
- The drag force exerted by air and liquids at slow speeds can be modeled as  $\vec{F}_{ac}^{d} = -b\vec{v}$ , where *b* is called the *damping coefficient*.
- In presence of a drag, the equation of motion becomes

$$m\frac{d^2x}{dt^2} + b\frac{dx}{dt} + kx = 0$$

• The solution of this equation takes the forms

$$x(t) = Ae^{-bt/2m} \sin(\omega_{d}t + \phi_{i})$$
$$\omega_{d} = \sqrt{\frac{k}{m} - \frac{b^{2}}{4m^{2}}} = \sqrt{\omega^{2} - \left(\frac{b}{2m}\right)^{2}}$$

• The figure shows oscillations for various values of the damping coefficient *b*.



- The ratio m/b has units of time and is called the time constant: τ ≡ m/b.
- Amplitude of the damped oscillation at any given time is given by

$$x_{\max}(t) = A e^{-t/2\tau}.$$

• The mechanical energy of the oscillator can be expressed as

$$E(t) = \frac{1}{2}m\omega^2 x_{\max}^2 = (\frac{1}{2}m\omega^2 A^2)e^{-t/\tau} = E_0 e^{-t/\tau}$$

## **Concepts: Fundamental characteristics of periodic motion**

- **Periodic motion** is any motion that repeats itself at regular time intervals. **Oscillation** (or **vibration**) is back-and-forth periodic motion.
- The **period** *T* is the minimum time interval in which periodic motion repeats, and the **amplitude** *A* of the motion is the magnitude of the maximum displacement of the moving object from its equilibrium position.

## **Chapter 15: Summary**

## **Quantitative Tools: Fundamental characteristics of periodic motion**

• The **frequency** *f* of periodic motion is the number of cycles per second and is defined as

$$f \equiv \frac{1}{T}$$

• The SI unit of frequency is the hertz (Hz):  $1 \text{ Hz} \equiv 1 \text{ s}^{-1}.$ 

### **Concepts: Simple harmonic motion**

• Simple harmonic motion is periodic motion in which the displacement of a system from its equilibrium position varies sinusoidally with time. A system moving in this way is called a *simple harmonic oscillator*.

### **Concepts: Simple harmonic motion**

 A restoring force that is linearly proportional to displacement tends to return a simple harmonic oscillator to its equilibrium position. For small displacements, restoring forces are generally proportional to the displacement and therefore cause objects to execute simple harmonic motion about any stable equilibrium position.

### **Concepts: Simple harmonic motion**

• A **phasor** is a rotating arrow whose component on a vertical axis traces out simple harmonic motion. The **reference circle** is the circle traced out by the tip of the phasor, and the length of the phasor is equal to the amplitude *A* of the simple harmonic motion.

# **Quantitative Tools: Simple harmonic motion**

The angular frequency ω of oscillation is equal to the rotational speed of a rotating phasor whose vertical component oscillates with a frequency *f*. Angular frequency is measured in s<sup>-1</sup> and is related to the frequency (measured in Hz) by

$$\omega = 2\pi f.$$

• For a simple harmonic oscillator of amplitude *A*, the displacement *x* as a function of time is

$$x(t) = A \sin(\omega t + \phi_{\rm i}),$$

where the sine argument is the **phase**  $\phi(t)$  of the periodic motion,

$$\phi(t)=\omega t+\phi_{\rm i},$$

and  $\phi_i$  is the *initial phase* at t = 0.

# **Quantitative Tools: Simple harmonic motion**

• The *x* components of the velocity and acceleration of a simple harmonic oscillator are

$$\upsilon_x \equiv \frac{dx}{dt} = \omega A \cos(\omega t + \phi_i)$$
$$a_x \equiv \frac{d^2 x}{dt^2} = -\omega^2 A \sin(\omega t + \phi_i).$$

## **Quantitative Tools: Simple harmonic motion**

• Any object that undergoes simple harmonic motion obeys the simple harmonic oscillator equation:

$$\frac{d^2x}{dt^2} = -\omega^2 x,$$

where *x* is the object's displacement from its equilibrium position.

• The mechanical energy *E* of an object of mass *m* undergoing simple harmonic motion is

$$E = \frac{1}{2}m\omega^2 A^2.$$

# **Concepts: Fourier series**

• Fourier's theorem says that any periodic function with period *T* can be written as a sum of sinusoidal simple harmonic functions of frequency  $f_n = n/T$ , where *n* is an integer. The n = 1 term is the fundamental frequency or first harmonic, and the other components are higher harmonics.

# **Chapter 15: Summary**

# **Quantitative Tools: Oscillating springs**

• For an object attached to a light spring of spring constant *k*, the simple harmonic oscillator equation takes the form

$$\frac{d^2x}{dt^2} = -\frac{k}{m}x,$$

and the angular frequency of the oscillation is

$$\omega = +\sqrt{\frac{k}{m}}.$$

• The motion of the object is described by

$$x(t) = A \sin\left(\sqrt{\frac{k}{m}} t + \phi_{i}\right).$$

# **Chapter 15: Summary**

#### **Concepts: Rotational oscillations**

- A horizontal disk suspended at its center by a thin fiber forms a type of *torsional oscillator*.
- A pendulum is any object that swings about a pivot. A *simple pendulum* consists of a small object (the bob) attached to a very light wire or rod.

### **Quantitative Tools: Rotational oscillations**

• If a torsional oscillator of rotational inertia *I* is twisted through a small angle from its equilibrium position  $\vartheta_0$  to position  $\vartheta$ , the restoring torque  $\tau_{\vartheta}$  is

$$\tau_{\vartheta} = -\kappa(\vartheta - \vartheta_0),$$

where  $\kappa$  is the *torsional constant*. When  $\vartheta_0 = 0$ , the simple harmonic oscillator equation for the torsional oscillator is

$$\frac{d^2\vartheta}{dt^2} = -\frac{\kappa}{I}\vartheta.$$

### **Quantitative Tools: Rotational oscillations**

• The rotational position  $\vartheta$  of the torsional oscillator at instant *t* is given by

$$\vartheta = \vartheta_{\max} \sin(\omega t + \phi_{i}),$$

where  $\vartheta_{\max}$  is the maximum rotational displacement and

$$\omega = \sqrt{\frac{\kappa}{I}}.$$

# **Chapter 15: Summary**

### **Quantitative Tools: Rotational oscillations**

• For small rotational displacements, the simple harmonic oscillator equation of a pendulum is

$$\frac{d^2\vartheta}{dt^2} = -\frac{m\ell_{\rm cm}g}{I}\vartheta$$

and its angular frequency is

$$\omega = \sqrt{\frac{m\ell_{\rm cm}g}{I}},$$

where  $\ell_{cm}$  is the distance from the center of mass of the pendulum to the pivot.

• The period of a simple pendulum is  $T = 2\pi \sqrt{\frac{\ell}{g}}$ .

# **Concepts: Damped oscillations**

- In **damped oscillation**, the amplitude decreases over time due to energy dissipation. The cause of the dissipation is a *damping force* due to friction, air drag, or water drag.
- A damped oscillator that has a large *quality factor Q* keeps oscillating for many periods.

### **Quantitative Tools: Damped oscillations**

• At low speeds, the damping force  $\vec{F}_{ao}^{d}$  tends to be proportional to the velocity of the object:

$$\vec{F}_{ao}^{d} = -b\vec{\upsilon},$$

where *b*, the *damping coefficient*, has units of kilograms per second.

• For small damping, the position x(t) of a damped spring is  $x(t) = Ae^{-bt/2m} \sin(\omega_d t + \phi_i)$ 

and its angular frequency  $\omega_d$  is

$$\omega_{\rm d} = \sqrt{\frac{k}{m} - \frac{b^2}{4m^2}} = \sqrt{\omega^2 - \left(\frac{b}{2m}\right)^2}.$$